



# **FRTN10 Multivariable Control, Lecture 12**

**Automatic Control LTH, 2017**

# Course Outline

- L1-L5 Specifications, models and loop-shaping by hand
- L6-L8 Limitations on achievable performance
- L9-L11 Controller optimization: Analytic approach
- L12-L14 Controller optimization: Numerical approach
  - 12 **Youla parameterization, internal model control**
  - 13 Synthesis by convex optimization
  - 14 Controller simplification

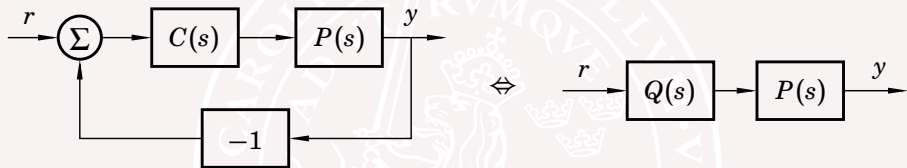
# Lecture 12 – Outline

- 1 The Quola parameterization
- 2 Internal model control (IMC)

[Glad&Ljung Section 8.4]

# Basic idea of Youla and IMC

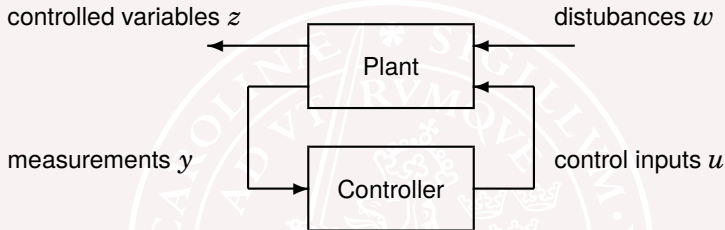
Assume stable SISO plant  $P$ . Model for design:



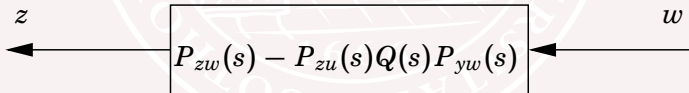
$$\frac{PC}{1 + PC} = PQ$$
$$Q = \frac{C}{1 + PC}$$

Design  $Q$  to get desired closed-loop properties. Then  $C = \frac{Q}{1 - QP}$

## General idea for Lectures 12–14



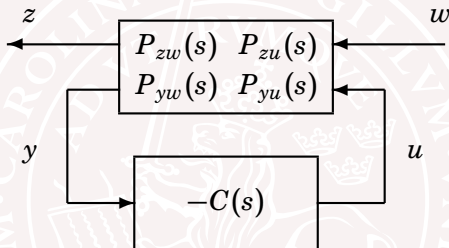
The choice of controller corresponds to designing a transfer matrix  $Q(s)$ , to get desirable properties of the following map from  $w$  to  $z$ :



Once  $Q(s)$  has been designed, the corresponding controller can be found.

# The Youla (Q) parameterization

General closed-loop control system:

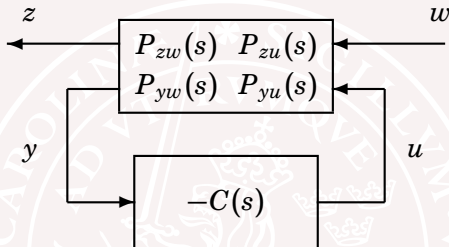


$$Z(s) = P_{zw}(s)W(s) + P_{zu}(s)U(s)$$

$$Y(s) = P_{yw}(s)W(s) + P_{yu}(s)U(s)$$

$$U(s) = -C(s)Y(s)$$

# The Youla (Q) parameterization



Closed-loop transfer function from  $w$  to  $z$ :

$$G_{zw}(s) = P_{zw}(s) - P_{zu}(s) \underbrace{C(s)[I + P_{yu}(s)C(s)]^{-1}}_{=Q(s)} P_{yw}(s)$$

Given  $Q(s)$ , the controller is  $C(s) = [I - Q(s)P_{yu}(s)]^{-1}Q(s)$

# All stabilizing controllers

Suppose the plant  $P = \begin{bmatrix} P_{zw} & P_{zu} \\ P_{yw} & P_{yu} \end{bmatrix}$  is stable. Then

- Stability of  $Q$  implies stability of  $P_{zw} - P_{zu}QP_{yw}$
- If  $Q = C[I + P_{yu}C]^{-1}$  is unstable, then the closed loop is unstable.

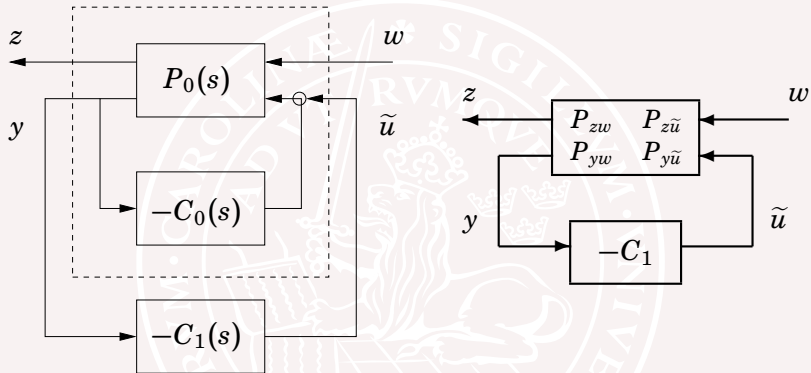
Hence, if  $P$  is stable then **all stabilizing controllers** are given by

$$C(s) = [I - Q(s)P_{yu}(s)]^{-1}Q(s)$$

where  $Q(s)$  is an arbitrary stable transfer function.

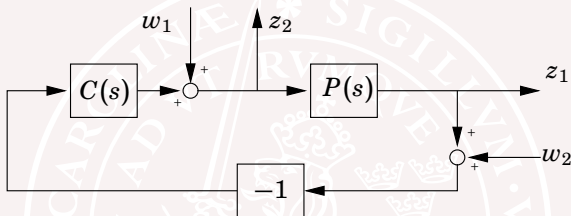


# Dealing with unstable plants



If  $P_0(s)$  is unstable, let  $C_0(s)$  be some stabilizing controller. Then the previous argument can be applied with  $P_{zw}$ ,  $P_{z\tilde{u}}$ ,  $P_{yw}$ , and  $P_{y\tilde{u}}$  representing the stabilized system.

## Example – DC-motor



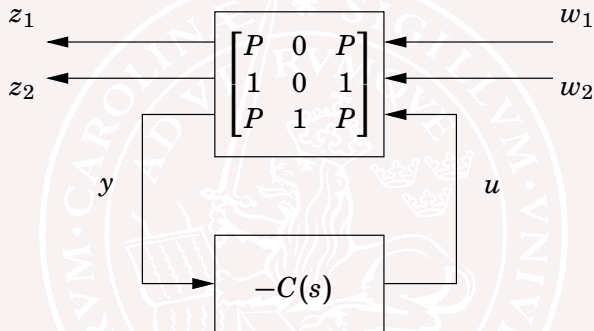
Assume we want to optimize the closed-loop transfer matrix from  $(w_1, w_2)^T$  to  $(z_1, z_2)^T$ ,

$$G_{zw}(s) = \begin{bmatrix} \frac{P}{1+PC} & \frac{-PC}{1+PC} \\ \frac{1}{1+PC} & \frac{-C}{1+PC} \end{bmatrix}$$

when  $P(s) = \frac{20}{s(s+1)}$ . How to parameterize all stabilizing controllers  $C(s)$ ?

# Stabilizing controller for DC-motor

Generalized plant model:

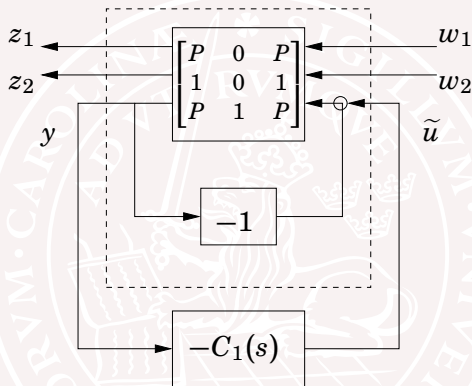


$P(s) = \frac{20}{s(s+1)}$  is not stable, so introduce

$$C(s) = C_0(s) + C_1(s)$$

where  $C_0(s) = 1$  stabilizes the plant;  $P_c(s) = \frac{P(s)}{1+P(s)} = \frac{20}{s^2+s+20}$

## Redrawn diagram for DC-motor example

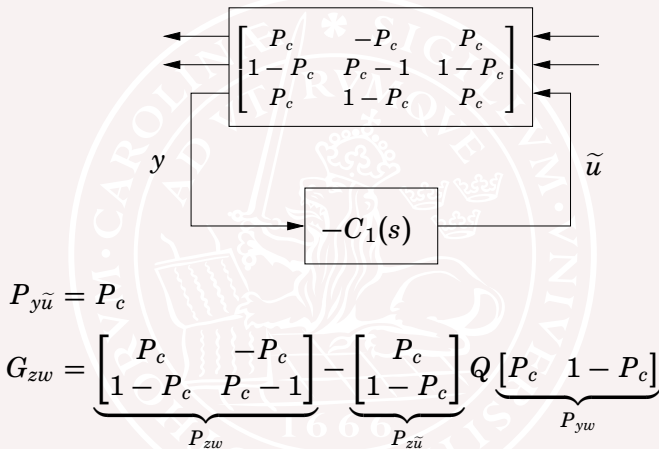


$$z_1 = Pw_1 + P(\tilde{u} - y)$$

$$z_2 = w_1 + \tilde{u} - y$$

$$y = Pw_1 + w_2 + P(\tilde{u} - y) \Rightarrow y = \frac{P}{1+P}w_1 + \frac{1}{1+P}w_2 + \frac{P}{1+P}\tilde{u}$$

## Redrawn diagram for DC-motor example



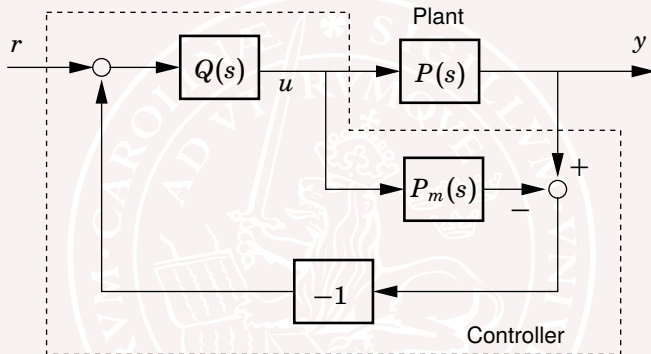
Apply optimization (Lec. 13) to find  $Q(s)$ . Then  $C(s) = 1 + \frac{Q(s)}{1 - Q(s)P_{y\tilde{u}}(s)}$

# Lecture 12 – Outline

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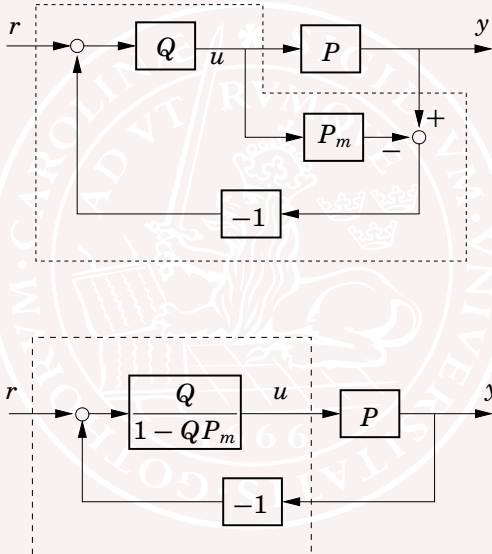
# Internal model control (IMC)



Feedback is used only if the real plant  $P(s)$  deviates from the model  $P_m(s)$ .  $Q(s)$ ,  $P(s)$ ,  $P_m(s)$  must be stable.

If  $P_m(s) = P(s)$ , the transfer function from  $r$  to  $y$  is  $P(s)Q(s)$ .

## Two equivalent diagrams





# IMC design rules

When  $P = P_m$ , the transfer function from  $r$  to  $y$  is  $P(s)Q(s)$ .

For perfect reference following, one would like to have  $Q(s) = P^{-1}(s)$ , but that is not possible (why?)

Design rules:

- If  $P(s)$  is strictly proper, the inverse would have more zeros than poles. Instead, one can choose

$$Q(s) = \frac{1}{(\lambda s + 1)^n} P^{-1}(s)$$

where  $n$  is large enough to make  $Q$  proper. The parameter  $\lambda$  determines the speed of the closed-loop system.

(cf. feedforward design in Lecture 4)

# IMC design rules

- If  $P(s)$  has unstable zeros, the inverse would be unstable. Options:
  - Remove every unstable factor  $(-\beta s + 1)$  from the plant numerator before inverting.
  - Replace every unstable factor  $(-\beta s + 1)$  with  $(\beta s + 1)$ . With this option, only the phase is modified, not the amplitude function.
- If  $P(s)$  includes a time delay, its inverse would have to predict the future. Instead, the time delay is removed before inverting.

# IMC design example 1 — first-order plant

$$P(s) = \frac{1}{\tau s + 1}$$

$$Q(s) = \frac{1}{\lambda s + 1} P(s)^{-1} = \frac{\tau s + 1}{\lambda s + 1}$$

$$C(s) = \frac{Q(s)}{1 - Q(s)P(s)} = \frac{\frac{\tau s + 1}{\lambda s + 1}}{1 - \frac{1}{\lambda s + 1}} = \underbrace{\frac{\tau}{\lambda} \left( 1 + \frac{1}{s\tau} \right)}_{\text{PI controller}}$$

Note that  $T_i = \tau$

This way of tuning a PI controller is known as *lambda tuning*

## IMC design example 2 — non-minimum phase plant

$$P(s) = \frac{-\beta s + 1}{\tau s + 1}$$

$$Q(s) = \frac{(-\beta s + 1)}{(\beta s + 1)} P(s)^{-1} = \frac{\tau s + 1}{\beta s + 1}$$

$$C(s) = \frac{Q(s)}{1 - Q(s)P(s)} = \frac{\frac{\tau s + 1}{\beta s + 1}}{1 - \frac{(-\beta s + 1)}{(\beta s + 1)}} = \underbrace{\frac{\tau}{2\beta} \left( 1 + \frac{1}{s\tau} \right)}_{\text{PI controller}}$$

Note that, again,  $T_i = \tau$

The gain is adjusted in accordance with the fundamental limitation imposed by the RHP zero in  $1/\beta$ .

# IMC design for dead-time processes

Consider the plant model

$$P_m = P_{0m}e^{-s\tau}$$

where the deadtime  $\tau$  is assumed known and constant.

Let  $C_0 = Q/(1 - QP_{0m})$  be a controller designed for the delay-free plant model  $P_{0m}$ . Then

$$Q = \frac{C_0}{1 + C_0P_{0m}}$$

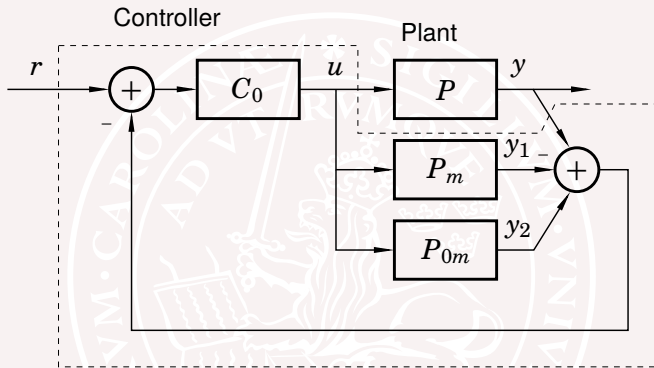
The rule of thumb tell us to use the same  $Q$  also for systems with delays.

This gives

$$C = \frac{Q}{1 - QP_{0m}e^{-s\tau}} = \frac{C_0}{1 + (1 - e^{-s\tau})C_0P_{0m}}$$

This modification of  $C_0$  to account for a time delay is known as a Smith predictor.

# Smith predictor



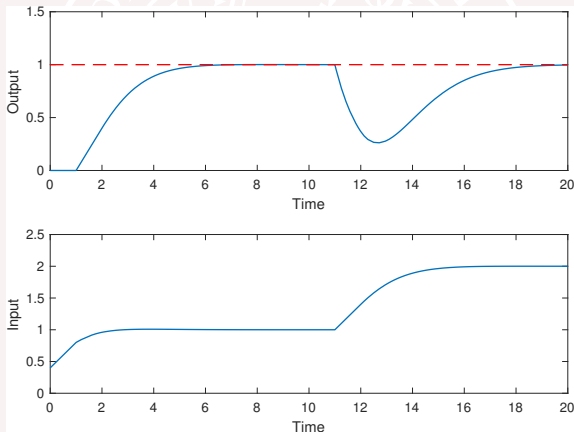
Ideally  $y$  and  $y_1$  cancel each other and only feedback from  $y_2$  “without delay” is used. If  $P = P_m$  then

$$Y(s) = \frac{C_0(s)P_{0m}(s)}{1 + C_0(s)P_{0m}(s)} e^{-s\tau} R(s)$$

## Example

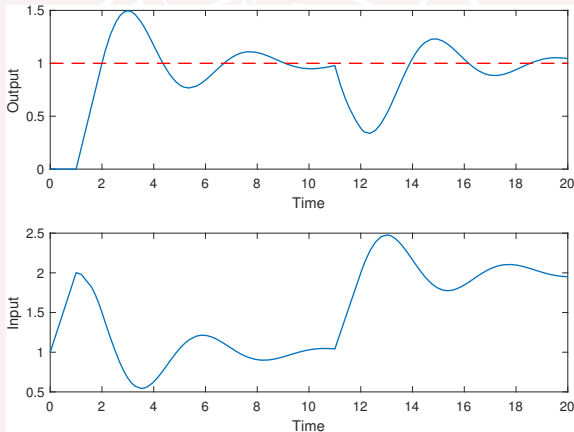
Plant:  $P(s) = \frac{1}{s+1}e^{-s}$ , nominal controller:  $C_0(s) = K\left(1 + \frac{1}{s}\right)$

Simulation with  $K = 0.4$ , no Smith predictor:



# Example

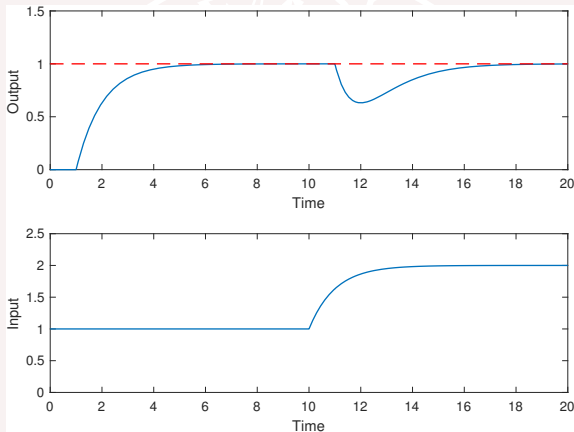
Simulation with  $K = 1$ , no Smith predictor:





# Example

Simulation with  $K = 1$  with Smith predictor:



(But do not forget the fundamental limitation imposed by the time delay!)

## Lecture 12 – summary

- Idea: Parameterize the closed loop as

$$G_{yr} = PQ \quad \text{SISO case, for IMC design}$$

or

$$G_{zw} = P_{zw} - P_{zu}QP_{yw} \quad \text{General MIMO case, suitable for optimization}$$

for some stable  $Q$ .

- After designing  $Q$ , the controller is given by

$$C = \frac{Q}{1 - QP} \quad \text{SISO case}$$

or

$$C = [I - QP_{yu}]^{-1}Q \quad \text{General MIMO case}$$