

Lecture 4:

Today's lecture: Loop shaping design

- Specifications in frequency domain
- Loop shaping design

Continuing from lecture 3...

- The closed-loop system
 - Look at all transfer functions in the loop! (Gang of Four / Gang of six)
 - Robustness

New today

- Loop shaping

[Glad & Ljung] Ch. 6.4–6.6, 8.1–8.2 + AK

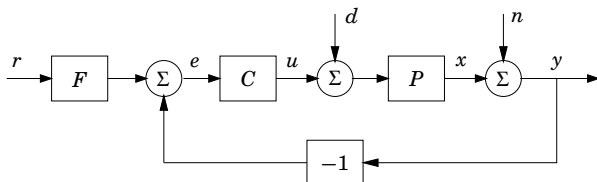
Designing System with Two Degrees of Freedom

Design procedure:

- Design the feedback C to achieve
 - Small sensitivity to load disturbances d
 - Low injection of measurement noise n
 - High robustness to process variations
- Then design the feedforward F to achieve desired response to command signals r

For many problems in process control the load disturbance response is much more important than the set point response. The set point response is more important in motion control. Few textbooks and papers show more than set point responses.

A Basic Control System

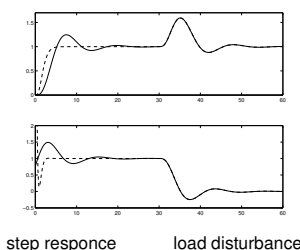


Ingredients:

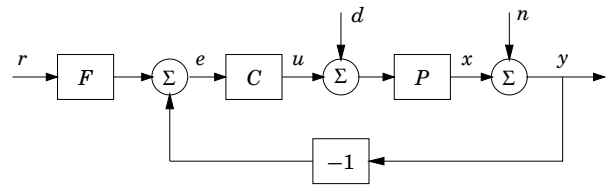
- Controller: feedback C , feedforward F
- Load disturbance d : Drives the system from desired state
- Measurement noise n : Corrupts information about x
- Process variable x should follow reference r

Time domain specifications

- Step response (w.r.t reference and/or load disturbance)
 - rise-time T_r
 - overshoot
 - settling time T_s
 - static error e_0
- ...



Relations between signals



$$\begin{aligned} X &= \frac{P}{1+PC}D - \frac{PC}{1+PC}N + \frac{PCF}{1+PC}R \\ Y &= \frac{P}{1+PC}D + \frac{1}{1+PC}N + \frac{PCF}{1+PC}R \\ U &= -\frac{PC}{1+PC}D - \frac{C}{1+PC}N + \frac{CF}{1+PC}R \end{aligned}$$

Gang of Four / Gang of Six

Six transfer functions are required to show the properties of a basic feedback loop. Four characterize the response to load disturbances and measurement noise.

$$\begin{array}{cc} \frac{PC}{1+PC} & \frac{P}{1+PC} \\ \frac{C}{1+PC} & \frac{1}{1+PC} \end{array}$$

Two more are required to describe the response to set point changes.

$$\frac{PCF}{1+PC} \quad \frac{CF}{1+PC}$$

Key Issues

Find a controller that

- A:** Reduces effects of load disturbances
- B:** Does not inject too much measurement noise into the system
- C:** Makes the closed loop insensitive to variations in the process
- D:** Makes output follow command signals

Convenient to use a controller with two degrees of freedom, i.e. separate signal transmission from y to u and from r to u . This gives a complete separation of the problem: Use feedback to deal with A, B, and C. Use feedforward to deal with D!

Frequency domain specifications

Closed loops specs.

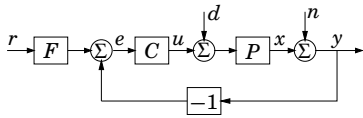
- resonance peak M_p
- bandwidth ω_B (see definition!!)

Open-loop measures

- M_S and M_T -circles
- Amplitude margin A_m , phase margin ϕ_m
- cross-over frequency ω_c
- ...

Note: Often the design is made in Bode/Nyquist/Nichols diagrams for loop-gain $L = PC$ (open loop system)

Specifications on closed loop system



Would like:

- ▶ Small influence of low-frequency disturbance d on z
- ▶ Limited amplification of high-frequency noise n in control u
- ▶ Robust stability despite high-frequency uncertainty

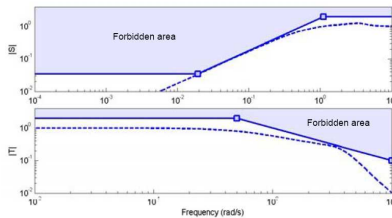
Frequency domain specs.

Closed-loop:

Find specifications W_T and W_S for closed-loops transfer functions s.t

$$\begin{aligned} |T(i\omega)| &\leq |W_T^{-1}(i\omega)| \\ |S(i\omega)| &\leq |W_S^{-1}(i\omega)| \end{aligned}$$

(Magnitude transfers to singular values for MIMO-systems)



Design: Consider open loop system

Try to look at **loop-gain** $L = PC$ for design and to translate specifications of S & T into specs of L

$$S = \frac{1}{1+L} \approx 1/L \quad \text{if } L \text{ is Large}$$

$$T = \frac{L}{1+L} \approx L \quad \text{if } L \text{ is small}$$

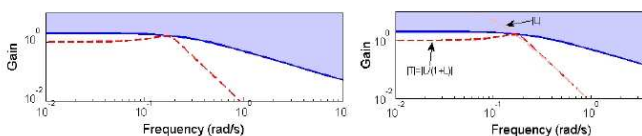
Classical loop shaping:

- ▶ design C so that $L = PC$ satisfies constraints on S and T
- ▶ how are the specifications related?
- ▶ what to do with the regions around cross-over frequency ω_c (where $|L| = 1$)?

Complementary Sensitivity vs Loop Gain

$$T = \frac{L}{1+L}$$

$$|T(i\omega)| \leq |W_T^{-1}(i\omega)| \Leftrightarrow \frac{|L(i\omega)|}{|1+L(i\omega)|} \leq |W_T^{-1}(i\omega)|$$



large frequencies, W_T^{-1} small $\Rightarrow |T| \approx |L|$

$$|L(i\omega)| \leq |W_T^{-1}(i\omega)| \quad (\text{approx.})$$

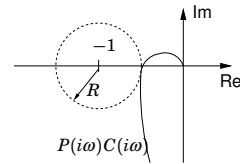
(typically valid for $\omega > \omega_{OT}$)

[Lecture 2]:

Different interpretations of the *Sensitivity function* $S = \frac{1}{1+PC}$

1. $S = G_{n \rightarrow y}(s) = G_{r \rightarrow e}(s)$ [See previous slide]
2. $S = \frac{d(\log H)}{d(\log P)} = \frac{dH/H}{dP/P}$ ("How sensitive is the closed loop system H wrt process variations")
3. S measures the distance from the Nyquist plot to $(-1+0i)$.

$$R^{-1} = \sup_{\omega} \left| \frac{1}{1+P(i\omega)C(i\omega)} \right|$$



These specifications can not be chosen independently of each other.

$$S + T = 1$$

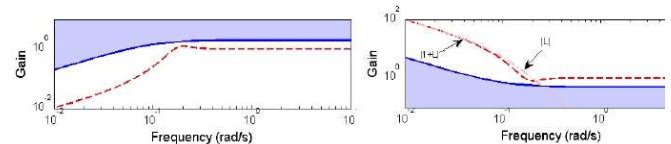
Limiting factors:

- ▶ Fundamental limitations [Lecture 7/Ch 7]:
 - ▶ RHP zero at $z \Rightarrow \omega_{BS} \leq z/2$
 - ▶ Time delay $T \Rightarrow \omega_{BS} \leq 1/T$
 - ▶ RHP pole at $p \Rightarrow \omega_{OT} \geq 2p$
- ▶ Bode's integral theorem
 - ▶ The "waterbed effect"
- ▶ Bode's relation
 - ▶ good phase margin requires certain distance between ω_{BS} and ω_{OT}
- ▶ Model uncertainty:
 - ▶ Robust stability gives new "forbidden area"
 - ▶ Robust performance somewhat more complicated

Sensitivity vs Loop Gain

$$S = \frac{1}{1+L}$$

$$|S(i\omega)| \leq |W_S^{-1}(i\omega)| \Leftrightarrow |1+L(i\omega)| > |W_S(i\omega)|$$

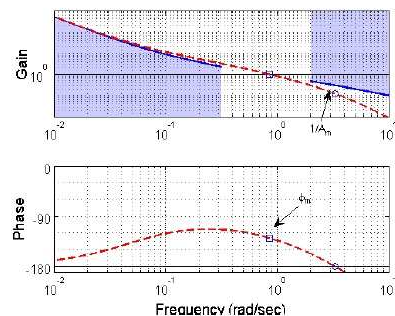


small frequencies, W_S large $\Rightarrow 1+L$ large, and $|L| \approx |1+L|$.

$$|L(i\omega)| \geq |W_S(i\omega)| \quad (\text{approx.})$$

(typically valid for $\omega < \omega_{BS}$)

Resulting constraints on loop-gain L :

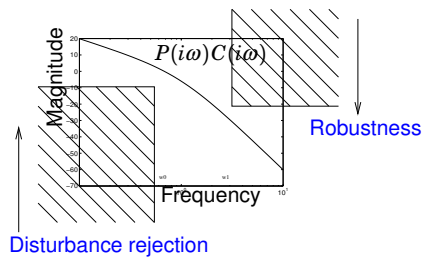


Remark: approximations inexact around cross-over frequency ω_c . In this region, focus is on stability margins A_m, ϕ_m .

These requirements is to say that the *loop transfer matrix*

$$L = P(i\omega)C(i\omega)$$

should have small norm $\|P(i\omega)C(i\omega)\|$ at high frequencies, while at low the frequencies instead $\|[P(i\omega)C(i\omega)]^{-1}\|$ should be small.



Classical loop shaping

Map specifications on requirements on loop gain L.

- ▶ Low-frequency specifications from W_S
- ▶ High-frequency specifications from W_T^{-1}
- ▶ Around cross-over frequency, mapping is crude
 - ▶ Position cross-over frequency (constrained by W_S , W_T)
 - ▶ Adjust phase margin (e.g. from M_S , M_T specifications)

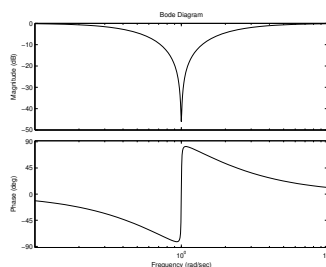
Properties of leads-lag elements

- ▶ Lag (phase retarding) elements
 - ▶ Reduces static error
 - ▶ Reduces stability margin
- ▶ Lead (phase advancing) elements
 - ▶ Increased speed by increased ω_c
 - ▶ Increased phase
 - ⇒ May improve stability
- ▶ Gain
 - ▶ Translates magnitude curve
 - ▶ Does not change phase curve

See "Collection of Formulae" for lead-lag link diagrams

Example of other compensation-link:

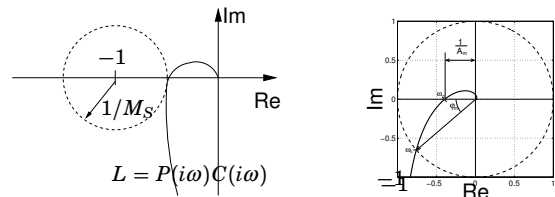
$$\text{Notch-filter } \frac{s^2 + 0.01s + 1}{s^2 + 2s + 1}$$



M_S and M_T and stability margins

Specifying $|T(i\omega)| \leq M_T$ and $|S(i\omega)| \leq M_S$ gives bounds for the amplitude and phase margins (but not the other way round!)

$$|S(i\omega)| \leq M_S \implies A_m > \frac{M_S}{M_S - 1}, \quad \phi_m > 2 \arcsin \frac{1}{M_S}$$



Q: Why does not A_m and ϕ_m give bounds on M_T and M_S ?

Lead-lag compensation

Shape loop gain $L = PC$ using a compensator C composed of

- ▶ Lag (phase retarding) elements

$$C_{lag} = \frac{s + a}{s + a/M}, \quad M > 1$$

- ▶ Lead (phase advancing) elements

$$C_{lead} = N \frac{s + b}{s + bN}, \quad N > 1$$

- ▶ Gain

$$K$$

Typically

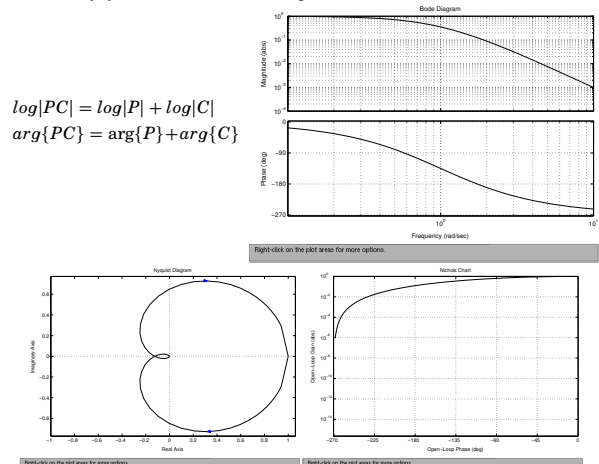
$$C = K \frac{s + a}{s + a/M} \cdot N \frac{s + b}{s + bN}$$

Iterative lead-lag design

- ▶ Step 1: Lag (phase retarding) element
 - ▶ Add phase retarding element to get low-frequency asymptote right
- ▶ Step 2: Phase advancing element
 - ▶ Use phase advancing element to obtain correct phase margin
- ▶ Step 3: Adjust gain
 - ▶ Usually need to "lift up" or "push down" amplitude curve to obtain the desired cross-over frequency.

Adjusting the gain in Step 3 leaves the phase unaffected, but may ruin low-frequency asymptote (need to revise lag element) ⇒ An iterative method!

Bode, Nyquist and Nichols diagrams



Qualitative Feedback Design Theory was developed by Horowitz *et. al.* to ensure desired loop-gain properties despite model uncertainties.

Basic principle: Let the (uncertain) system be represented by several transfer functions and at each frequency we get a corresponding set (template) of points which all should satisfy the constraints.

Equivalently

$$F(s) \approx \frac{1 + P(s)C(s)}{P(s)C(s)}$$

Exact equality is generally impossible because of pole excess in P .

The simplest and most common approximation is to use a constant gain

$$F = \frac{1 + P(0)C(0)}{P(0)C(0)}$$

Example

$$P(s) = \frac{1}{(s+1)^4} \quad F(s) = \frac{1 + P(s)C(s)}{P(s)C(s)(sT+1)^d}$$

The closed loop transfer function from r to u then becomes

$$\frac{C(s)}{1 + P(s)C(s)} F(s) = \frac{(s+1)^4}{(sT+1)^4}$$

which has low-fq gain 1, but gain $1/T^4$ for $\omega \rightarrow \infty$.

Design of Feedforward revisited

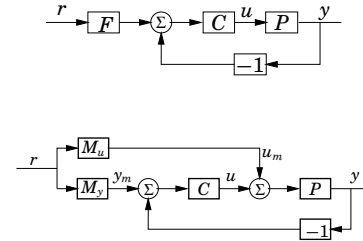
The transfer function from r to $e = y_m - y$ is $(M_y - PM_u)S$

Ideally, M_u should satisfy $M_u = M_y/P$. This condition does not depend on C !

Since $M_u = M_y/P$ should be stable, causal and not include derivatives we find that

- ▶ Unstable process zeros must be zeros of M_y
- ▶ Time delays of the process must be time delays of M_y
- ▶ The pole excess of M_y must be greater than the pole excess of P

Take process limitations into account!



The reference signal r specifies the desired value of y .

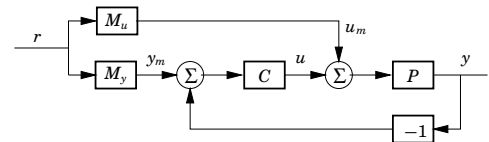
Ideally

$$\frac{P(s)C(s)}{1 + P(s)C(s)} F(s) \approx 1$$

A more advanced option is

$$F(s) = \frac{1 + P(s)C(s)}{P(s)C(s)(sT+1)^d}$$

for some suitable time constant T and d large enough to make F proper and implementable.



Notice that M_u and M_y can be viewed as generators of the desired output y_m and the inputs u_m which corresponds to y_m .

Example of Feedforward Design revisited

If

$$P(s) = \frac{1}{(s+1)^4} \quad M_y(s) = \frac{1}{(sT+1)^4}$$

then

$$M_u(s) = \frac{M_y(s)}{P(s)} = \frac{(s+1)^4}{(sT+1)^4} \quad \frac{M_u(\infty)}{M_u(0)} = \frac{1}{T^4}$$

Fast response (T small) requires high gain of M_u .

Bounds on the control signal limit how fast response we can obtain.

Frequency design;

- ▶ Good mapping between S, T and $L = PC$ at low and high frequencies (mapping around cross-over frequency less clear)
- ▶ Simple relation between C and $L \Rightarrow$ easy to shape L !
- ▶ Lead-lag control: iterative adjustment procedure
- ▶ What if closed-loop specifications are not satisfied?
 - ▶ we made a poor design (did not iterate enough), or
 - ▶ the specifications are not feasible (fundamental limitations in Lecture 7)
- ▶ Alternatives:
 - ▶ H_∞ -optimal control: finds stabilizing controller that satisfies constraints, if such a controller exists

Feedforward design

Case study DVD-player

- ▶ Use **loop-shaping techniques from this lecture** for focus control design in DVD-player
- ▶ track following (modelling of disturbances, control)

